

## WIRE STEERING

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### ABSTRACT

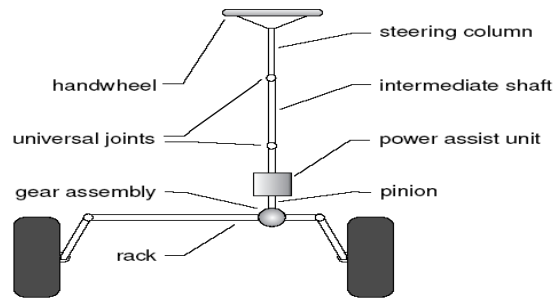
*The automotive industry has already implemented many advanced computer systems in an attempt to increase safety and comfort of drivers. In parallel with these advancements we see a big shift from mechanical systems to electrical systems and steer-by-wire is another implementation that is very promising in terms of safety and functionality. Already, there are some commercial prototypes of such 'by-wire' systems [1] and there is a lot of research, both academic [2] and commercial [3], in the field. For my Engineering Senior Design Project at Swarthmore College, I chose to work on a steer-by-wire system to gain more insight into control theory and I thought the double-control system that provided the crucial feedback to the driver was an interesting engineering problem.*

### I. INTRODUCTION

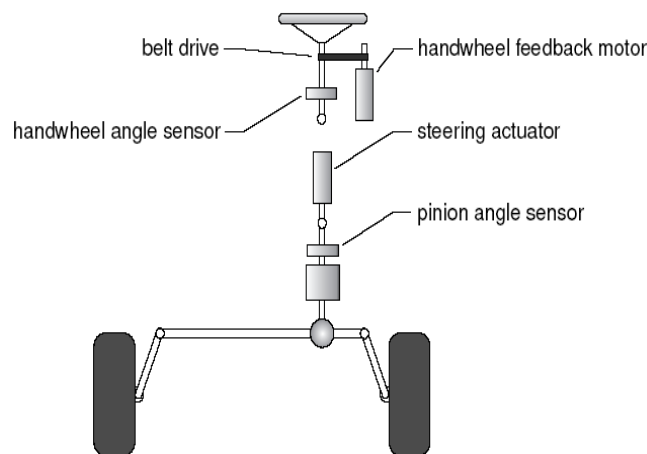
Unlike the conventional steering system where a hand-operated steering wheel is used to turn the front wheels through the steering column, steer-by-wire technology removes the mechanical and physical links between the driver (steering wheel) and the front wheels, and replace them with electronic actuators and other components. A steer-by-wire system aims to eliminate the physical connection between the steering wheel and the wheels of a car by using electrically controlled motors to change the direction of the wheels and to provide feedback to the driver. Today's automobiles benefit more and more from the many uses of electronic systems. The integration of a steer-by-wire system can enhance these systems in many ways. In particular, the handling and the safety of the cars can be improved significantly. Since a steer-by-wire system is easily modifiable, different drivers will be able to adjust the system to accommodate their styles and this will enhance handling. In addition, disabled people and the elderly will benefit immensely from steer-by-wire because they will be able to situate the steering wheel to meet special needs. Traction control systems are very closely tied with driving safety and they can be enhanced with steer-by-wire vastly. For instance, in a situation where the car starts over steering (when the rear of the vehicle heads towards the outside of the corner), the natural instinct of many inexperienced drivers is to turn the steering wheel towards the inside, which in turn causes more over steer. A steer-by-wire system could be modified to take control in a situation like this to steer to the outside. Since there are virtually no physical connections between the steering wheel and the wheels, a steer-by-wire system can be implemented on different cars easily. The steering wheel could be replaced on either side of a car (or anywhere else). Both of these improvements would reduce costs of production and allow a wider range of designs. The downsides of a steer-by-wire system are maintenance and power cost. Conceivably steer by-wire will use more power than the currently used system, however considering the power consumption of power steering the power cost will be insignificant. There might also be more electrical failures, but presumably steer-by-wire systems will last longer

because they have fewer mechanical parts and will improve safety and therefore help the overall maintenance costs.

## II.DESIGN



**Conventional Steering System**



**Steer-by-Wire System**

### 2.1. System Overview

The steer-by-wire system consists of two main parts. The steering section consists of the steering wheel, the feedback actuator and the feedback actuator angle sensor. The wheel section contains the wheels, the rack and pinion, a steering actuator and the pinion angle sensor. Figure 2 shows the system components. In my system I only demonstrated the double control mechanism and did not implement it in a rack and pinion configuration. The feedback angle sensor provides the steering actuator with its primary input signal and the pinion angle sensor provides the feedback motor primary signal. The small size of the feedback motor lets the driver rotate the steering wheel with little difficulty. As soon as the driver starts steering, the control mechanism tries to push the steering wheel back into place (and the wheels into the position dictated by the current position of the steering wheel) and this mimics the resistive force of a real steering wheel. However, changing the proportional constant of the feedback motor can make it harder/easier for the driver to steer and allows for adjustable steering

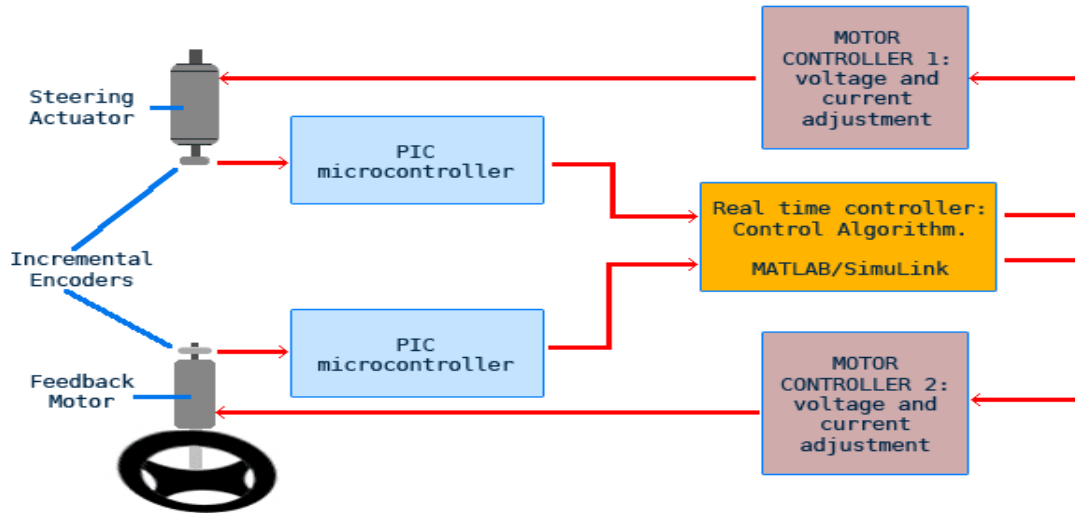


Figure 1: Overview of system

### 3.1.1 Physical Components

#### 3.1.1.1 Steering Actuator

The steering actuator needs to be very powerful in order to turn the wheels of a car when the car is loaded. Minimizing the effects of unwanted disturbances also requires a powerful motor. In the design, I wanted to use a small wagon as a model and measurements showed that a motor with a torque of about 80lb-in was necessary. Ideally this motor would be a brushless DC motor in order to reduce noise and maximize motor life. However, the high cost and low availability of brushless motors led me to acquire the DC motor shown in Figure 3. This is a Groschopp 50757, a 88.9 lb-in

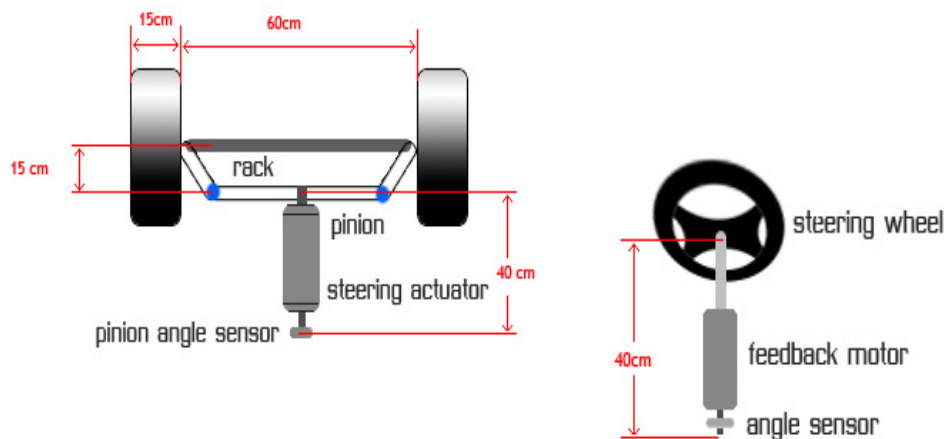


Figure 2: Components of a steer-by-wire system

motor that uses a 12V source and draws approximately 3.4 amps. Initially, I wanted to use a small wagon to implement my project and show that I could steer the wagon with a person seated on it. In order to find the right size motor, I put a 200lb weight located in the center of the wagon body and used a 39.5 inch crank to turn the rack of the wagon. With the wheels on a painted floor the force required was about 1.9lbs (and it never exceeded 2lbs). Thus the total torque was measured to be  $39.5 \cdot 2 = 80$ lbs. Although this motor has sufficient torque, the maximum rotation speed is limited to 13.3 RPM. Using only 13.3 RPM, it is impossible to simulate rapid movements experienced in a car, therefore this motor requires gears to increase rotational speed at the expense of reducing force.



Figure 3: Steering Actuator

The high current characteristics of this motor make it impossible to control using configuration and require a special controller. The specification sheet for this motor can be found in the appendix .g an H-bridge

### 3.1.1.2 Feedback Motor



Figure 4: Feedback Motor

The feedback motor does not have to be as powerful as the steering actuator. In fact, it has to be much less powerful in order to be turned easily by a driver. I opted to use the motor seen in Figure 4. It was a used motor I acquired from the department. Initially I tested this motor with full power, and the torque it provided was similar to the torque felt in a real car and I concluded that this would be a sufficient motor. The power requirements were also reasonable. It operates at 12V and draws less than 1.5A, so the low current requirements allow for an H-Bridge controller

### 3.1.1.3 Angular Sensors

The angular sensors of the system are very crucial and they need to be very accurate because little perturbations or errors ultimately make the control of the system much harder for a driver. In a real implementation of a steer-by-wire system then would have to be very high sensitivity and accuracy in order to minimize risks. In my project

I used two optical digital encoders that were used in a previous project. These are BEI Duncan's EX-11 and MX-15 encoders and both of the sensors are seen in Figure 5.

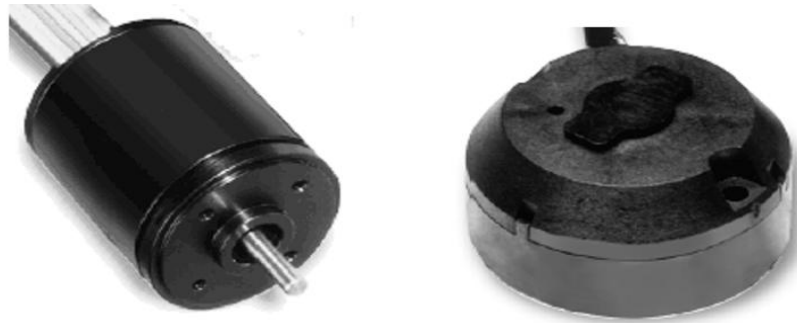


Figure 5: Angular sensors: Incremental encoders

Optical digital encoders' precision and accuracy make them preferable over potentiometers. They output 1024 pulses per revolution in two channels. The channels have a 90-degree offset to indicate direction of rotation. Figure 6 shows the output wave form of the encoders. The specifications of both encoders can be found in the appendix.

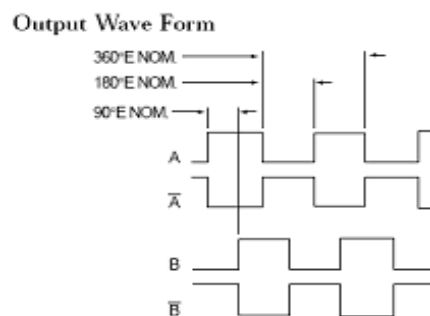


Figure 6: Output wave form of the encoders (taken from the specifications)

### 3.1.1.4 Microcontrollers

The rotary sensors have four possible combinations of outputs and these outputs have to be processed in order to measure the actual angle of rotation. This is done using the PIC microcontrollers. Once the C code for PICs is compiled, the ICD interface lets us install the program onto the PIC and the EEPROM technology allows the program to stay on the PIC even when the power is turned off and on. You can see a picture of the PIC microcontroller in Figure 7, and see the schematics of the PIC in Figure 8. I appropriated C code from a project done by Emery Ku to run the PIC controllers. The code can be found in the appendix. Since the PIC is a digital device, it only outputs 0 or 5V. In order to get the intermediate values, we have to use the pulse-width-modulated output of the PICs. This is done by coupling the output with a resistor and a capacitor. Depending on the values of the capacitor and resistor, the noise and speed of the output can be varied. I opted for a 12k resistor and a 33 $\mu$ F capacitor.

### 3.1.1.5 Electronics

Although most of the control mechanism is done in software, I needed some electronics that provide discernible inputs for the controller and regulated the output from the computer to drive the motors. Firstly, since the rotary encoders did not provide absolute angle information, I used PIC microcontrollers to convert line counts to a voltage in the range of 0-5V. Secondly, the output of the digital acquisition board attached to the computer

could generate voltages from -5 to +5 volts with a resolution of 1.22 mV and the current output is too low to drive the motors used. Although I never measured the maximum current output, I believe it is in the milliamp range. At least it is essentially zero compared to 3.4A needed by the steering actuator. Figure 1 shows where the electronics are located in the system.

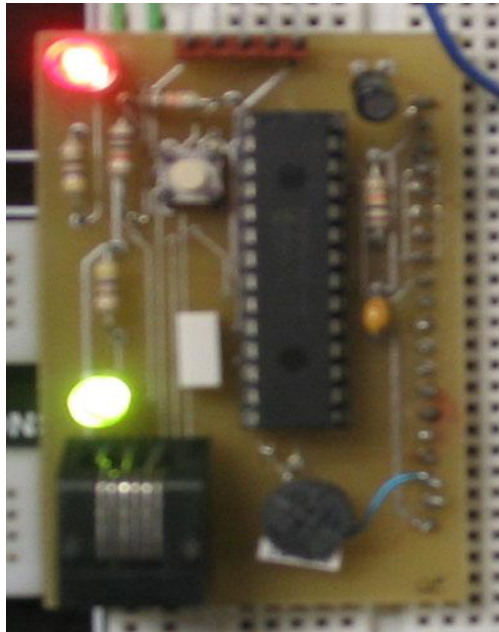


Figure7:PICmicrocontroller

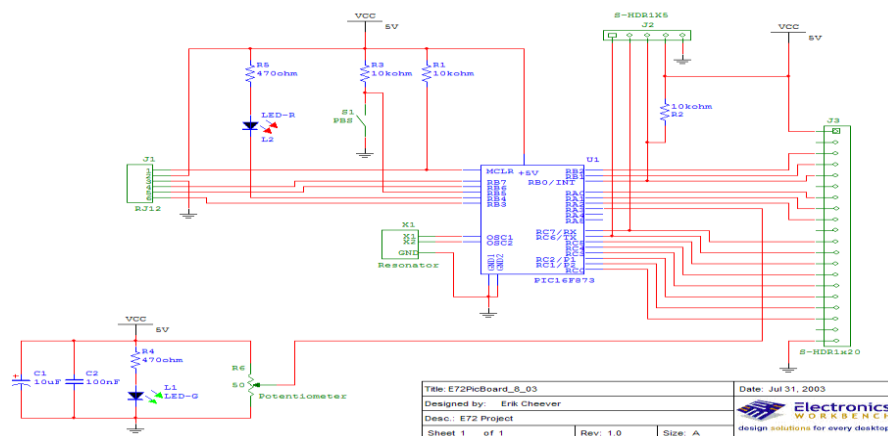
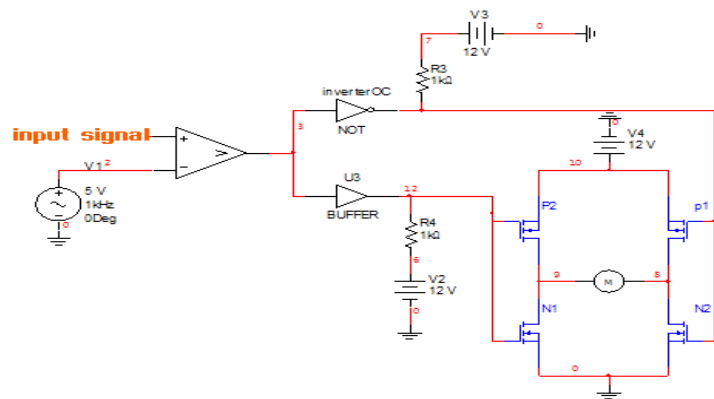


Figure 8: PIC microcontroller schematics (courtesy of Erik Cheever).

### 3.1.1.6 Pulse Width Modulation and the H-Bridge

The feedback motor requires low current and this makes the H-bridge configuration coupled with a pulse width modulated input signal (seen in Figure 9) a good choice for a controller. The pulse width-modulation uses a comparator to compare a triangular wave to a user-specified input. The output is a square wave that has a duty cycle proportional to the specified input. The square wave is very useful because it can be used to switch transistors on and off. Figure 9:





**Figure 9: Pulse width modulation and the H-Bridge Configuration**

The pulse-width modulated signal goes through an inverter (4017) and a buffer 4016) to adjust the voltage to a range of 0-12V instead of the comparator output range of 0–5V. The outputs of the buffer and inverter are adjusted using a resistor between the outputs and a 12V source. Using the same source for the H-bridge and the output adjustment is a good idea because that way the gate and drain voltages of the mosfets are adjusted accordingly. The H-Bridge takes its name from the way it looks (see Figure 9). It consists of pairs of P-channel and N-channel mosfets. The right side and the left side of the H-bridge get exactly opposite signals and since the pulse-width-modulated signal runs the gate voltages, we get a very quick switching effect through the motor. Note that at a given time we only have P1 and N1 (which are on opposite sides) or P2 and N2 in operation. The current direction keeps changing and the motor stays idle when the duty cycle is 50%. When the duty cycle increases (or decreases), it allows more current to flow though the motor in one direction than the other and thus the motor starts rotation. One drawback of the h-bridge is that it dissipates most power when the motor is idle. If we consider the fact that most of the time the steering wheel is idle the h-bridge can be very inefficient in terms of power consumption. However, for our purposes it gets the job done.

### 3.1.1.7 Motor Controller for Steering Actuator

As mentioned earlier, the steering actuator is very powerful and draws a lot of current. Therefore a powerful controller is necessary to control it. I chose the KBBC-24M (see Figure 10) because it can be modified easily for use in various motors and can be used in future projects. With the wig-wag option selected, KBBC-24M accepts an input in the 0–5V range where 0 and 5V represent the fastest reverse and forward speeds respectively.



**Figure 10: KBBC-24M controller for the steering actuator**

### 3.1.1.8 Simulink Modeling

Most of the modeling and control is done using SimuLink and Matlab . The Real-Time Workshop in Simulink allows the user to design the controller in blocks and then compiles the scheme and drums the program in real-time as long as the computer is on. As opposed to past years, when the DAQ Board could only be used for a limited number of data samples (and therefore time), the Real-Time Workshop is a huge improvement. In the steer-by-wire system, I implement the model seen in Figure 11. Although the controls of both motors seem similar, their inherent differences require different control parameters. The input to both controllers is governed by the difference in the rotary sensor readings. In other words, the error signal is the difference in the angular sensor voltages. Both systems use PID controllers but the parameters vary. Also, since integration causes more noise, the integration constant is very low. Because the steering wheel angle sensor and the pinion angle sensor will usually be very close to each other, the integral control component does not help the system very much. In Figure 11, you can see that the output of the PID goes through a saturation and a constant of 2.5 is added after the saturation. This is because both motors are at rest when the inputs are 2.5V and they work in the range of 0–5V. The saturation adjusts the voltage to a range of -2.5–2.5V and the addition of 2.5V puts the final controller output in a range of 0–5V.

### 3.1.1.9 Computer Modeling

Although the control system can be done in hardware, it is much easier to implement the control system in software. I chose to use Matlab because I was familiar with it from previous classes. There were some simulations of steer-by-wire systems done using Matlab[4] so I knew it would be a great choice.

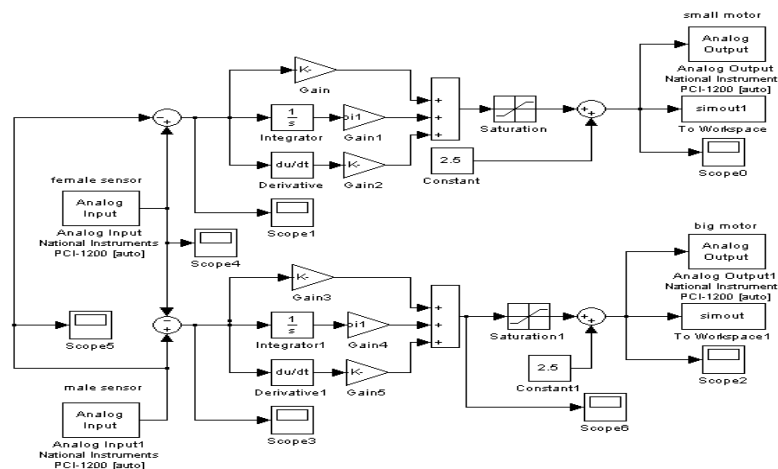


Figure 11: Control model

## IV. TESTING

### 4.1 Testing Procedure

Once I implemented my system with all individual parts working, I had to find optimal constants for the PID controller and adjust the saturation levels of the PID output. First I tested each motor individually, and observed their behavior for step inputs. I saw that the steering motor was very slow and that the controller output would be at a maximum (or minimum for reverse) most of the time. On the other hand, the feedback actuator would spin very fast with controller outputs of 5V or 0V, so I adjusted the saturation levels to -0.75–0.75V. With this saturation level, the input to the H-bridge was between 1.75V and 3.25V. This still provided enough feedback



and a fast response. Since the controller output was saturated most of the time, I did not do simulations on the computer, and instead I found the PID constants by trial and error. However this did not mean that I would have to rely on my luck to find the right values. Since I was familiar with the effects of changing control parameters, I could reach good values pretty soon and build on them.

#### 4.2 Determining PID constants

There are a few tuning methods to find the constants (or get in the ballpark of good constants). I first used the Ziegler-Nichols method introduced by John G. Ziegler and Nathaniel B. Nichols. This method starts with finding a critical gain  $K_c$  (the gain for which a proportional controller using this gain starts to oscillate). Then at the oscillation frequency it finds the oscillation period,  $P_c$ . Then the Ziegler-Nichols method suggests the values seen in Table 1.

Parameter	$K_p$	$K_i$	$K_d$
Value	$0.6K_c$	$P_c/2$	$P_c/8$

Table 1: PID controller constants using the Ziegler-Nichols method

Initially, this gave me some guidance, and I had some success with controlling each of the motors individually, but I noticed that when I put the two systems together, these parameters did not help me much. The feedback actuator would vibrate a lot especially because the steering motor was very slow in reacting. I noticed that if I held the steering rod in place and waited for the steering motor to approach its desired position, the integral control would increase the output to the feedback actuator too much, and releasing the steering rod would cause it to overshoot a lot. When the system was in equilibrium, small differences in the sensor angles caused vibrations in the feedback actuator so eventually, I opted to eliminate the integral component of the feedback motor. The steering motor controller was very slow in switching from maximum forward to maximum reverse direction, so the control of the motor was very hard. When I was deciding the PID constants, I noticed that all combinations pretty much gave similar results. In the end, I decided to use values for which the step response would cause the motor controller to switch in time to drive the motor in the opposite direction and stop it. In general I followed the guideline in Table 2 to achieve acceptable controller constants. The final values I used can be found in Table 3

Parameter	Rise Time	Overshoot	Settling Time
$K_p$	decrease	increase	small change
$K_i$	decrease	increase	increase
$K_d$	small change	decrease	decrease

Table 2: Effects of increasing parameters: a guideline for picking PID controller constants

	steering motor	feedback actuator
Kp	0.5	0.65
Ki	0.05	0
Kd	0.8	0.4

Table 3: PID controller values for the final design

## V. ADVANTAGES

1. No steering column – Simplify the design of a car’s interior, giving the driver more space as well as better safety in case of a crash (no intrusion of the steering column).
2. The absence of steering shaft and gear reduction mechanism allows much better utilization of the engine’s compartment.
3. Decreases the total weight of the car issuing better energy reduction effectiveness.
4. Easier implementation of left or right-hand driving.
5. No noise or vibration can reach the driver’s hands.
6. The most significant benefit is the ability to electronically augment the driver’s steering input depending of drive’s conditions, also called active steering.

## VI. LIMITATIONS

Pneumatic trail, a function of slip angle, is linear for small angles

Non-linearity problem for bigger angles

Linearization of friction in steering block

## VII. DISCUSSION

Although there were many problems I encountered with each part of the project, in the end, I was able to show that a steer-by-wire system could work. However, given the slow responses of the system, my setup is far from ideal and needs many improvements. The feedback motor worked very well and to a certain extent it simulated the real driving experience. The steering motor didn’t work very well at all, and this can be attributed to the characteristics of the motor and the motor controller. The motor controller had a very slow switching time. It would take about 0.4 seconds to switch from one direction to the other and this really hindered the operation. Also, the steering motor was quite powerful so even though the motor controller were able to switch very fast, the motor would still be slow to react. In a real implementation, it might be a good idea to use two dedicated motors, one for each direction. The steering wheel can also be improved by adding dampers and springs. I believe that with various improvements, steer-by-wire can achieve some success. Sometimes even very small delays in reaction can be fatal so the delay between steering wheel rotation and actual steering might pose a great threat in the development of SBW systems. However, given other advancements in automotive technology, we see that more and more AI systems are being integrated into cars and combining SBW systems with these can be very promising. Doing this project has improved my understanding of control systems greatly and I have gained

significant experience in combining electronic and mechanical systems. Although I have not accomplished my initial goals of the project 100%, I believe that there was a lot of progress made and I think that this system can be improved upon.

### **VIII. FUTURE WORK**

The SBW system I build can be improved a lot, but the main problem seems to be with the choice of controllers and motors. For a future project, given better equipment, this system could be implemented in a small model car and can be used for control theory demonstrations. New control systems, such as state-space controls, can be implemented to enhance the performance of the system. Although not in the near future, given enough resources, this system can be implemented in real road cars and perhaps be combined with regular steering to take advantage of the safety benefits of a steer-by-wire system.

### **ACKNOWLEDGEMENTS**

I would like to thank my advisor for his invaluable help with the project. Without his guidance, patience and encouragement, this project would never be complete. I would also like to thank Prof. Fred Orthlieb, Grant Smith, Ed Jaoudi, Aron Dobos, Danielle Miller and Emily Kan with their guidance and help with mechanical parts of the project. I would like to thank Prof Bruce Maxwell and Prof Cheever for all the engineering classes I took with them (a lot). Finally, I would like to thank all fellow Swarthmore Engineers for being the people they are.

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